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## Interface Element for Propagation in Layered Media

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# Interface Element for Propagation in Layered Media

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**SYNOPSIS** An interface element which accounts for the relative displacement between two layers through its stiffness properties and loss of energy through its damping properties has been developed. Its behaviour under static, free vibration, transient load and wave propagation in layered media has been studied.

## INTRODUCTION

A seismic wave travelling in layered media encounters interfaces at which energy may partly be transmitted through elastic bonding while some of the energy gets dissipated in the relative motion between the layers. The usual practice is to treat interface as perfectly welded/free, which represents extremities of the problem between which the actual situation may lie and has to be provided for. The loss of energy at the interface has also to be considered. The problem of relative displacements between two layers has been tackled by Goodman et al (1968) and Schafer (1975) by using bond (for interface) element.

## FORMULATION

In the interface element the relative displacement between two layers which is a measure of strain, is accounted for through its stiffness properties and loss of energy through its damping properties (Gadginglajkar, 1980).

For the plane stress/strain case, Fig. 1 shows an isoparametric interface element connecting two 8-noded isoparametric elements. The relative displacements  $\Delta u$  and  $\Delta v$  at a point are given by

$$\Delta u = u^A - u^B = \sum_{i=1}^3 N_i (u_i^A - u_i^B) \quad (1)$$

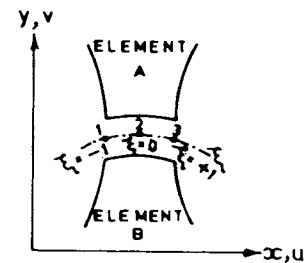
$$= \sum_{i=1}^3 N_i \Delta u_i \quad (2)$$

$$\text{Similarly } \Delta v = \sum_{i=1}^3 N_i \Delta v_i \quad (3)$$

The strains  $\underline{\epsilon}$  (curly line below a letter indicates a column matrix) are given by

$$\underline{\epsilon} = \begin{Bmatrix} \Delta u \\ \Delta v \end{Bmatrix} = \underline{B} \underline{\Delta} \quad (4)$$

where  $\underline{\Delta}$  is the nodal displacement vector. (Line below a letter indicates rectangular



$$N_i = \frac{1}{2} \xi \xi_i (1 + \xi \xi_i) \text{ FOR } i = 1, 3$$

$$N_i = (1 - \xi^2) \text{ FOR } i = 2$$

SHAPE FUNCTIONS

FIG. 1 INTERFACE ELEMENT FOR PLANE STRESS / STRAIN

matrix). The strains in global directions are transformed to local directions using usual transformation matrix  $\underline{T}$ .

$$\underline{\epsilon}' = \begin{Bmatrix} \Delta u' \\ \Delta v' \end{Bmatrix} = \underline{T} \begin{Bmatrix} \Delta u \\ \Delta v \end{Bmatrix} \quad (5)$$

Noting that  $\xi$  and  $\eta$  are orthogonal, it can be shown that

$$ds = |J| d\xi \quad (6)$$

and

$$\underline{T} = \frac{1}{|J|} \begin{bmatrix} \frac{\partial x}{\partial \xi} & \frac{\partial y}{\partial \xi} \\ \frac{\partial y}{\partial \xi} & \frac{\partial x}{\partial \xi} \end{bmatrix} \quad (7)$$

where  $J$  is the Jacobian matrix. Using Eqns. (4) and (5) one gets

$$\underline{\epsilon}' = \underline{T} \underline{B} \underline{\Delta} \quad (8)$$

The stress-strain relationship is given by

$$\underline{\sigma} = \begin{Bmatrix} \sigma'_s \\ \sigma'_n \end{Bmatrix} = \begin{bmatrix} k_s & 0 \\ 0 & k_n \end{bmatrix} \begin{Bmatrix} \Delta u' \\ \Delta v' \end{Bmatrix} = \underline{D} \underline{\epsilon}' \quad (9)$$

where  $k_s$  and  $k_n$  are tangential and normal stiffness parameters. The element stiffness

matrix  $\underline{k}$ , now, is obtained as

$$\underline{k} = \int_{-1}^1 \underline{B}^T \underline{T}^T \underline{D} \underline{T} \underline{B} |J| d\xi \quad (10)$$

(Superscript T indicates transpose of a matrix.)  
 $\underline{k}$  is integrated numerically using 3-point Gauss quadrature rule.

The damping force  $\underline{f}$  is assumed as

$$\underline{f} = \begin{bmatrix} c_s & c_n \\ 0 & c_n \end{bmatrix} \begin{Bmatrix} \dot{\Delta u}' \\ \dot{\Delta v}' \end{Bmatrix} = \underline{\mu} \dot{\underline{\epsilon}}' \quad (11)$$

where  $c_s$  and  $c_n$  are the tangential and normal damping parameters. (Dot on top indicates differentiation with respect to time.)  
 Considering the work lost in damping over the element and the work done by the nodal damping forces, the element damping matrix  $\underline{c}$  is given by

$$\underline{c} = \int_{-1}^1 \underline{B}^T \underline{T}^T \underline{\mu} \underline{T} \underline{B} |J| d\xi \quad (12)$$

For the axisymmetric case the corresponding element matrices are

$$\underline{k} = 2\pi \int_{-1}^1 \underline{B}^T \underline{T}^T \underline{D} \underline{T} \underline{B} r |J| d\xi \quad (13)$$

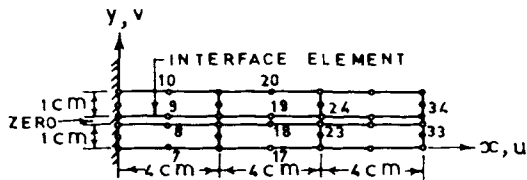
$$\underline{c} = 2\pi \int_{-1}^1 \underline{B}^T \underline{T}^T \underline{\mu} \underline{T} \underline{B} r |J| d\xi \quad (14)$$

where  $r$  is the radius.

Following the above steps and with appropriate modifications to displacement functions an infinite interface element is derived, Gadhinglajkar (1980).

## NUMERICAL STUDIES

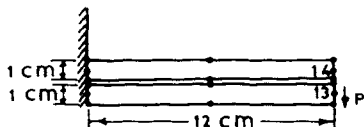
Numerical studies for stresses and deflections on cantilever with interface element subjected to static tip loads have been made. Table 1 gives influence of  $k_s$  and  $k_n$  on free vibration of the cantilever shown in Fig. 2.



$$E = 2 \times 10^6 \text{ kg/cm}^2, \quad \nu = 0.25$$

$$\eta = 0.8 \times 10^{-5} \text{ kg sec}^2/\text{cm}^4$$

FIG.2 CANTILEVER WITH INTERFACE ELEMENT



$$P = 100 \sin(NN/40) \pi, \quad \Delta t = 0.3 \mu\text{sec}, \quad NN = \text{NO. OF STEP}$$

$$E = 2 \times 10^6 \text{ kg/cm}^2, \quad \nu = 0.25, \quad \eta = 0.8 \times 10^{-5} \text{ kg sec}^2/\text{cm}^4$$

FIG.3 CANTILEVER WITH INTERFACE ELEMENT

In Table 2 are presented results of influence of  $c_s$  on displacement of the cantilever, Fig. 3. Comparison of displacements with  $c_s = 0.657 \times 10^{-1} \text{ kg-sec/cm}$ ,  $c_s = 6.57 \text{ kg-sec/cm}$  has been made, with those obtained for material damping assumed proportional to mass and stiffness, corresponding to the extreme frequencies of  $0.307 \times 10^4 \text{ rad/sec}$  and  $0.992 \times 10^7 \text{ rad/sec}$  of the cantilever, Fig. 3, and 5% of the critical damping.

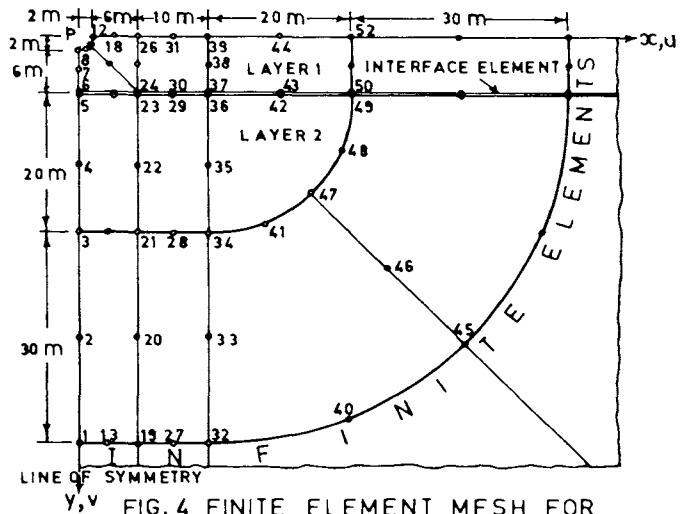
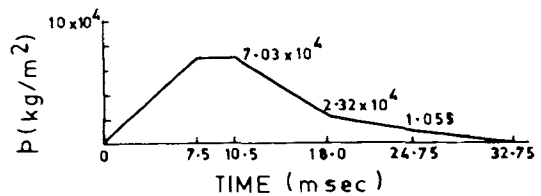


FIG.4 FINITE ELEMENT MESH FOR LAYERED MEDIA-SOURCE ON SURFACE



$$E_1 = 0.8 \times 10^9 \text{ kg/m}^2, \quad E_2 = 3 \times 10^9 \text{ kg/m}^2$$

$$\eta = 228.6 \text{ kg sec}^2/\text{m}^4, \quad \nu = 0.25$$

FIG.5 LOAD PULSE FOR LAYERED MEDIA

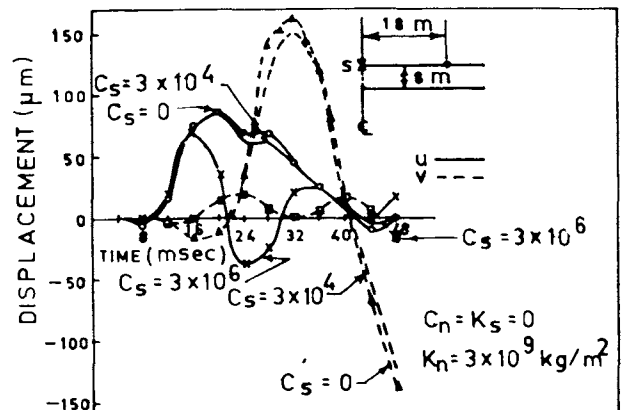


FIG.6 EFFECT OF  $C_s$  ON DISPLACEMENT AT 18 m ALONG FREE SURFACE

TABLE I. Effect of  $k_s$  and  $k_n$  on Frequency of Cantilever with Interface Element  
Frequency in rad/sec.

Type	Mode →	1	2	3	4	5	Last
i) Solid (without joint element)		$0.681 \times 10^4$ ( $0.7042 \times 10^4$ ) *	$0.37 \times 10^5$	$0.654 \times 10^5$	$0.886 \times 10^5$	$0.181 \times 10^6$	$0.346 \times 10^7$ (48th)
ii) With joint element $k_s = k_n = 10^6$		$0.574 \times 10^4$	$0.302 \times 10^5$	$0.596 \times 10^5$	$0.695 \times 10^5$	$0.159 \times 10^6$	$0.295 \times 10^7$ (60th)
iii) With joint element $k_s = k_n = 10^8$		$0.58 \times 10^4$	$0.317 \times 10^5$	$0.597 \times 10^5$	$0.756 \times 10^5$	$0.173 \times 10^6$	$0.809 \times 10^7$ (60th)
* Fundamental Frequency for solid cantilever (Warburton, 1976)							

TABLE II. Influence of  $c_s$  on displacements of Cantilever with Interface Element  
 $t = 0.3 \mu\text{sec}$  Displacements (cm) after 80 time steps

$c_s$		Node 13		Node 14	
		u	v	u	v
1	$c_s = 0.0$	$0.2139 \times 10^{-4}$	$-0.1618 \times 10^{-3}$	$-0.213 \times 10^{-3}$	$-0.1618 \times 10^{-3}$
2	$c_s = 0.657 \times 10^{-4}$	$0.2139 \times 10^{-4}$	$-0.1618 \times 10^{-3}$	$-0.213 \times 10^{-4}$	$-0.1618 \times 10^{-3}$
3	$c_s = 0.657 \times 10^{-2}$	$0.2121 \times 10^{-4}$	$-0.1618 \times 10^{-3}$	$-0.2112 \times 10^{-4}$	$-0.1618 \times 10^{-4}$
4	$c_s = 0.657 \times 10^{-1}$	$0.1966 \times 10^{-4}$	$-0.1614 \times 10^{-3}$	$-0.1958 \times 10^{-4}$	$-0.1614 \times 10^{-4}$
5	$c_s = 0.657 \times 10^1$	$0.1923 \times 10^{-5}$	$-0.1560 \times 10^{-3}$	$-0.1836 \times 10^{-5}$	$-0.156 \times 10^{-3}$
6	$c_s = 0.657 \times 10^3$	$0.6379 \times 10^{-7}$	$-0.155 \times 10^{-3}$	$0.2389 \times 10^{-7}$	$-0.155 \times 10^{-3}$
$k_s = 0, k_n = 10^8, c_n = 0$					

TABLE III. Comparison of Displacements for Different Values of  $c_s$  and  $\alpha$  and  $\beta$  for Cantilever with Interface Element.  $t = 0.3 \mu\text{sec}$ ; Displacements in cm

		Node 13						Node 14					
Time step		u	(c)	(a)	v	(c)		u	(c)	(a)	v	(c)	
	(a)	(b)			(b)		(a)	(b)			(b)		(c)
20	$0.2881 \times 10^{-5}$	$0.6877 \times 10^{-6}$	$0.2982 \times 10^{-5}$	$-0.232 \times 10^{-4}$	$-0.2305 \times 10^{-4}$	$-0.2318 \times 10^{-4}$	$-0.2804 \times 10^{-5}$	$-0.6110 \times 10^{-6}$	$-0.2905 \times 10^{-5}$	$-0.2286 \times 10^{-4}$	$-0.2271 \times 10^{-4}$	$-0.2284 \times 10^{-4}$	
40	$0.1478 \times 10^{-4}$	$0.3264 \times 10^{-5}$	$0.1532 \times 10^{-4}$	$-0.8275 \times 10^{-4}$	$-0.8118 \times 10^{-4}$	$-0.8270 \times 10^{-4}$	$-0.1471 \times 10^{-4}$	$-0.3196 \times 10^{-5}$	$-0.1525 \times 10^{-4}$	$-0.8274 \times 10^{-4}$	$-0.8117 \times 10^{-4}$	$-0.8270 \times 10^{-4}$	
60	$0.2325 \times 10^{-4}$	$0.4029 \times 10^{-5}$	$0.2431 \times 10^{-4}$	$-0.1403 \times 10^{-3}$	$-0.136 \times 10^{-3}$	$-0.1403 \times 10^{-3}$	$-0.2348 \times 10^{-4}$	$-0.4259 \times 10^{-5}$	$-0.2454 \times 10^{-4}$	$-0.1406 \times 10^{-3}$	$-0.1363 \times 10^{-3}$	$-0.1406 \times 10^{-3}$	
80	$0.1966 \times 10^{-4}$	$0.1923 \times 10^{-5}$	$0.2132 \times 10^{-4}$	$-0.1614 \times 10^{-3}$	$-0.156 \times 10^{-3}$	$-0.1613 \times 10^{-3}$	$-0.1958 \times 10^{-4}$	$-0.1836 \times 10^{-5}$	$-0.2124 \times 10^{-4}$	$-0.1614 \times 10^{-3}$	$-0.156 \times 10^{-3}$	$-0.1613 \times 10^{-3}$	
(a) $c_s = 0.657 \times 10^{-1}, c_n = 0, \alpha = \beta$ . (b) $c_s = 0.657 \times 10^{-1}, c_n = 0, \alpha = \beta$ . (c) $c_s = 0 = c_n, \alpha = 307, \beta = 1.01 \times 10^{-8}$													

Numerical studies have also been made to study the effect of consistent and lumped damping. Lumped damping matrix is obtained by scaling the diagonal terms of the latter so as to preserve the total element damping. Figs. 4 and 5 show finite element mesh and load pulse

applied in small cavity for the wave propagation in two layered media. Effect of  $c_s$  on displacements at 18 m along the free surface has been shown in Fig. 6. Influence of  $c_s$  and  $k_s$  on the particle motion diagrams at nodes 36 and 37 has been shown in Fig. 7.

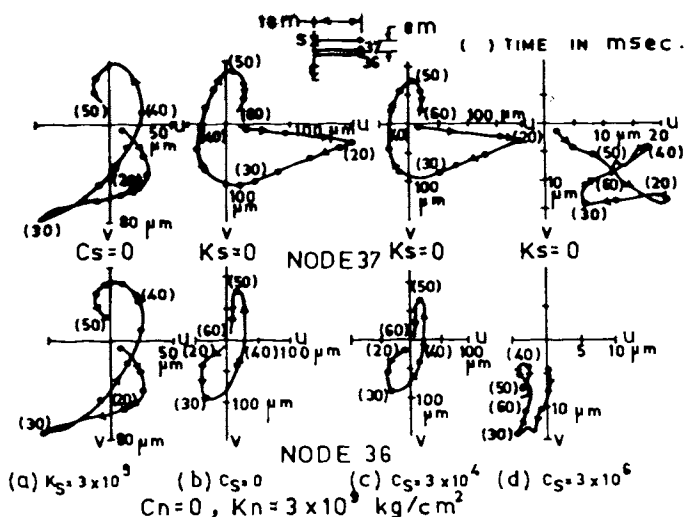


FIG. 7 PARTICLE DISPLACEMENT AT NODES 36 AND 37—EFFECT OF  $k_s$  AND  $c_s$

#### DISCUSSIONS AND CONCLUSIONS

To highlight the applicability of the interface element a simple cantilever has been studied. Influence of  $k_s$  and  $k_n$  on vertical tip deflection and fibre stresses under static load is studied. Results for the extreme cases of bonded and perfectly smooth cases showed good agreement with the alternative solutions. It is also observed that the modelling of cantilever by 3 elements across the span (total 9 elements) and 1 element across the span (total 3 elements) does not make appreciable difference in tip deflection.

It is seen from the results of free vibration of the cantilever, Table I, that for  $k_s = k_n = 10^8 \text{ kg/cm}^2$  the frequencies are higher compared to those for  $k_s = k_n = 10^6 \text{ kg/cm}^2$ ; the difference, being small in the initial modes, gradually increases in the higher modes. This is so because with the higher stiffness values, the higher modes will get more and more separated from those obtained with the lower stiffness values.

The values of  $k_s$  and  $k_n$  in the range of the value of  $E$  (the modulus of elasticity) correspond to perfect bonding through interface.

In the case of cantilever subjected to sinusoidal load, for want of adequate data the values of  $c_s$  and  $c_n$  are assumed to be related to critical damping for the media material. It is seen from Table II that significant change in vertical displacement occurs for  $c_s$  varied from  $0.657 \times 10^{-1} \text{ kg-sec/cm}$  to  $0.657 \times 10^1 \text{ kg-sec/cm}$ . The vertical displacements for the three cases shown in Table III, do not differ appreciably, the values in (a) and (c) being practically the same; while the horizontal displacements in (a) and (c) are closer and those for (b) differ considerably. The value of  $c_s$  in (a) may, therefore, be considered equivalent to material damping in (c) in the range of time considered.

Comparison of lateral tip deflection of the cantilever for the cases of consistent and lumped damping showed insignificant difference in the results.

In wave propagation in two layered media,

Fig. 4, to study the influence of  $c_s$  on displacements, initial value of  $c_s$  has been taken as  $1.971 \text{ kg-sec/m}$  on the presumption that the effect of  $c_s$  will be apparent for

$$\frac{c_s}{k_n} = \frac{0.657 \times 10^{-1}}{10^8} \text{ as has been the case with the}$$

cantilever. However no impact of  $c_s$  on displacements has been felt. Hence the values of  $c_s$  tried have been 3, 30,  $3 \times 10^2$ ,  $3 \times 10^3$ ,  $3 \times 10^6 \text{ kg-sec/m}$ . Upto  $3 \times 10^3 \text{ kg-sec/m}$  no change in displacements has been observed; at  $3 \times 10^4 \text{ kg-sec/m}$  the change in response becomes apparent and at  $3 \times 10^6 \text{ kg-sec/m}$  the response is dampened almost entirely and loses its oscillatory character significantly. Fig. 6 shows that with  $c_s = 3 \times 10^4 \text{ kg-sec/m}$  displacements get reduced marginally compared to those with  $c_s = 0$ , indicating that a small amount of energy is absorbed at the interface.

Influence of  $k_s$  on particle motion diagram, has been shown in Fig. 7. Because of nearly perfect bonding at interface due to  $k_s$ , the displacements on both sides of the interface show identical pattern. Comparison of influence of  $c_s$  on the particle motion at a typical pair of nodes on interface, Fig. 7, shows that for  $c_s = 0$  and  $c_s = 3 \times 10^4 \text{ kg-sec/m}$  the particle motion diagrams at the nodes are similar. It is also observed that with  $c_s = 3 \times 10^4 \text{ kg-sec/m}$  there is reduction in horizontal and vertical displacements. However reduction in horizontal displacements, as is expected, is more than that in the vertical component, because  $c_s$  is horizontal in this case. For  $c_s = 3 \times 10^6 \text{ kg-sec/m}$  the vertical displacements are not affected appreciably but because of the high value of  $c_s$ , the horizontal displacements are reduced and are also out of phase.

#### ACKNOWLEDGEMENT

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